

# Integrated Modeling Methodology Validation Using the Micro-Precision Interferometer Testbed

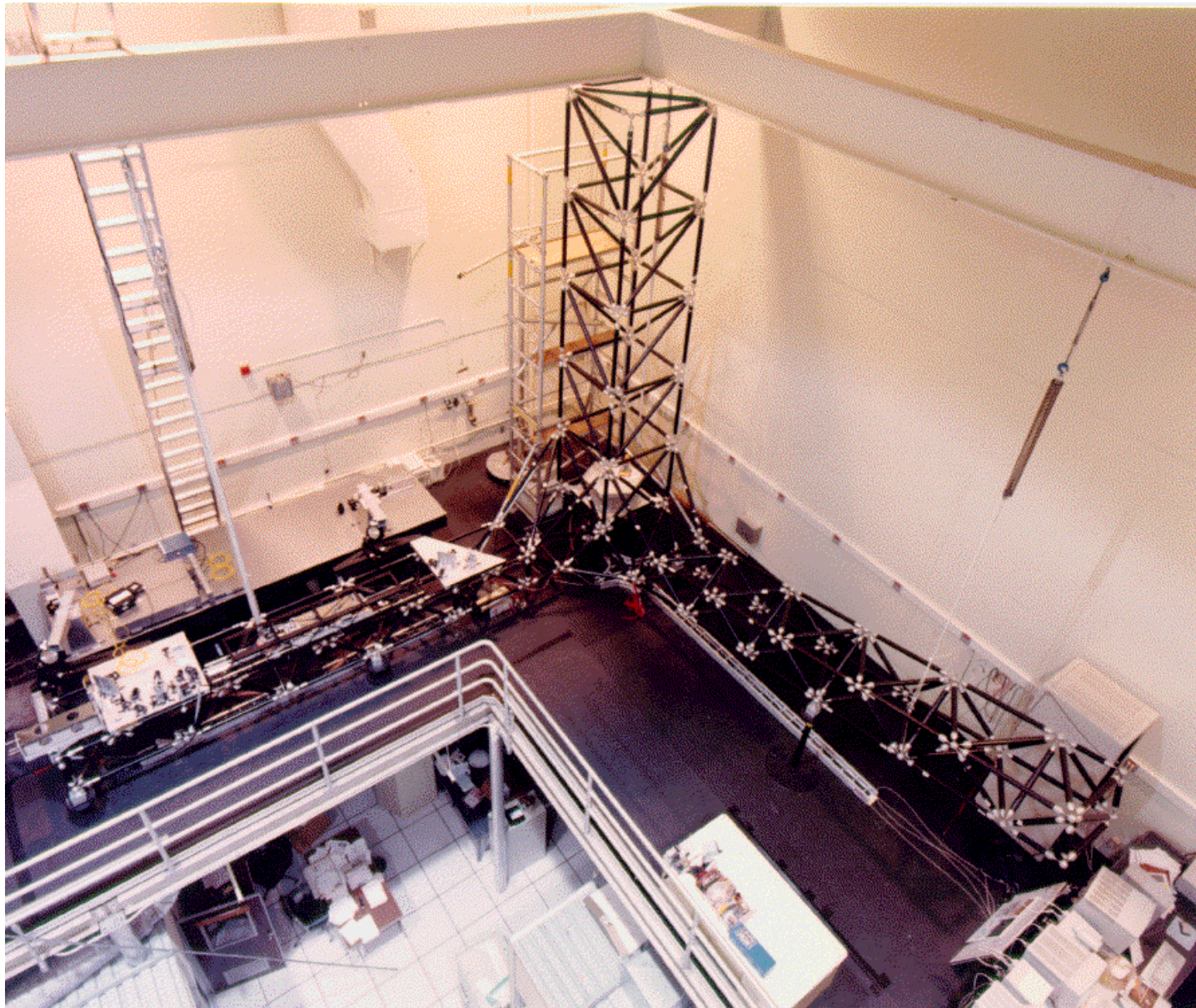
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- Integrated modeling description and tools
- Micro-precision interferometer testbed
- MPI integrated model
- MPI testbed measurements
- Validation metric
- Results

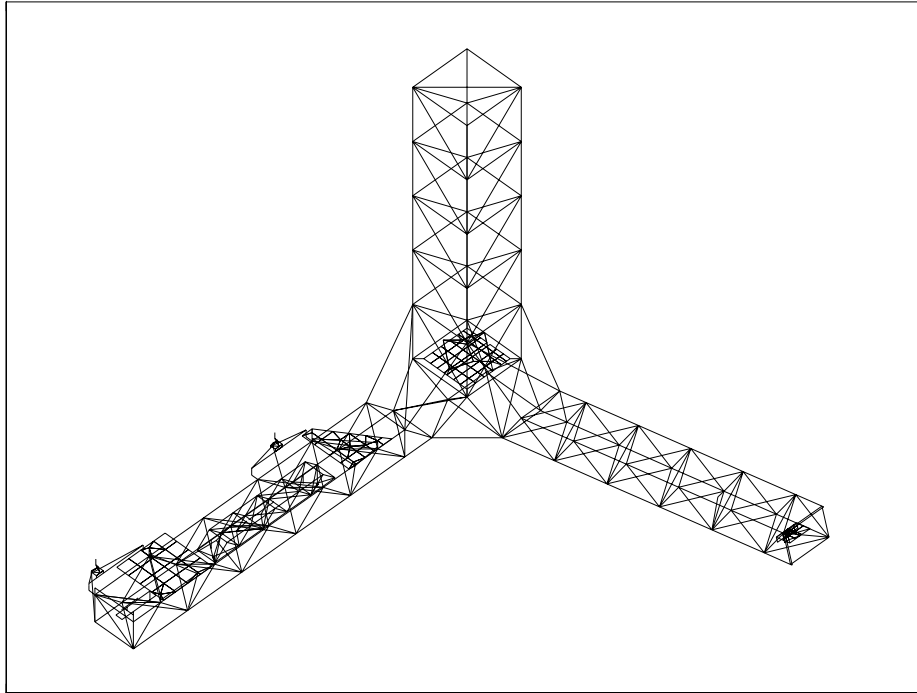
- Integrated modeling refers to modeling of controls, optics, and structures in a uniform software environment.
- Integrated modeling enables true multi-disciplinary:
  - Analysis
  - Design
  - Optimization
  - Diagnosis
- Integrated modeling is essential for spaceborne interferometry spacecraft and mission design:
  - Interferometer performance prediction in the presence of mechanical disturbances (*nanometer* stability requirements)
  - Requirements flow-down
  - Design trades

- Integrated Modeling of Optical Systems (IMOS) software package:
  - Matlab toolbox that enables structural and optical modeling
  - Includes functions for model integration
  - Utilizes plethora of Matlab controls and analysis functions
- Controlled Optics Modelling Package (COMP):
  - FORTRAN-compiled, stand-alone program for sophisticated optical modeling (e.g., diffraction and image synthesis)
  - Maintains compatibility with structural and controls models.
  - Interfaces easily with IMOS.
- IMOS and COMP have been used to evaluate *conceptual designs* of many interferometry missions: SIM, SONATA, OSI, POINTS, DLI, SITE, ISIS.
- Novel *modeling methodology* must be validated in order to have confidence in spacecraft and mission analyses.





## Finite Element Geometry



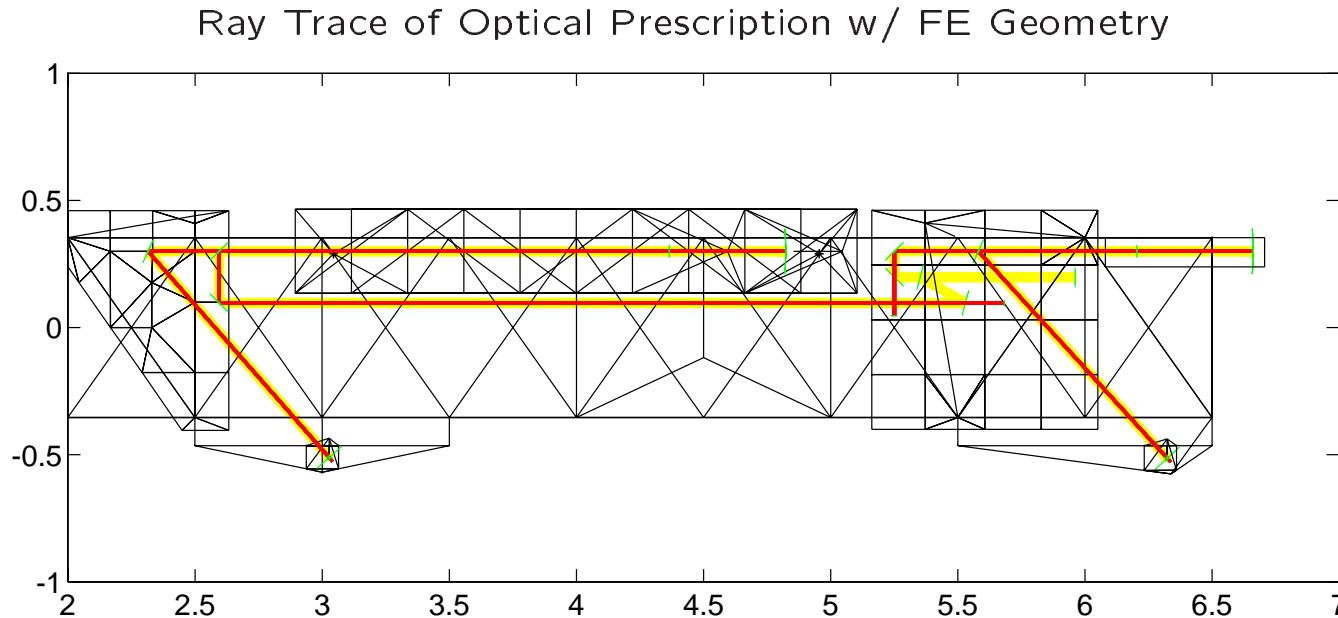
- Structural model specified in IMOS.
- Structural model consists of plate, beam, truss, and rigid body elements (RBEs).
- 2,577 total dofs: 1,832 independent w.r.t. multi-point constraints
- **Experimentally determined element properties** consistent with validation of modeling *methodology*.

- Finite element description ( $d \in R^{2577}$ ):

$$M\ddot{d} + Kd = B_f f$$

- Incorporation of multi-point constraints from RBEs ( $d_n \in R^{1832}$ ):

$$d = \begin{bmatrix} d_n \\ d_m \end{bmatrix} = Gd_n \Rightarrow M_{nn}\ddot{d}_n + K_{nn}d_n = B_{nf}f$$



- Optical prescription specifies shapes, positions, and orientations of optical elements.
- Prescription is specified in IMOS relative to the structural model, thereby easing model integration.
- Analytic differential ray trace (COMP) yields linear optical perturbation model:

$$y_{opt} = C_{opt} d$$

- Obtain eigensolution of FEM,  $(\Omega, \Phi_n)$ :

$$\begin{aligned}\ddot{\eta} + 2Z\Omega\dot{\eta} + \Omega^2\eta &= \Phi_n^T B_{nf} f \\ d &= G \Phi_n \eta\end{aligned}$$

with diagonal modal damping,  $Z$ , experimentally obtained from the testbed.

- Truncate modes above expected disturbance bandwidth (900 Hz), and convert to first-order model:

$$x = \begin{bmatrix} \eta_k \\ \dot{\eta}_k \end{bmatrix} \Rightarrow \begin{cases} \dot{x} = Ax + Bf \\ d = C_d x \end{cases}$$

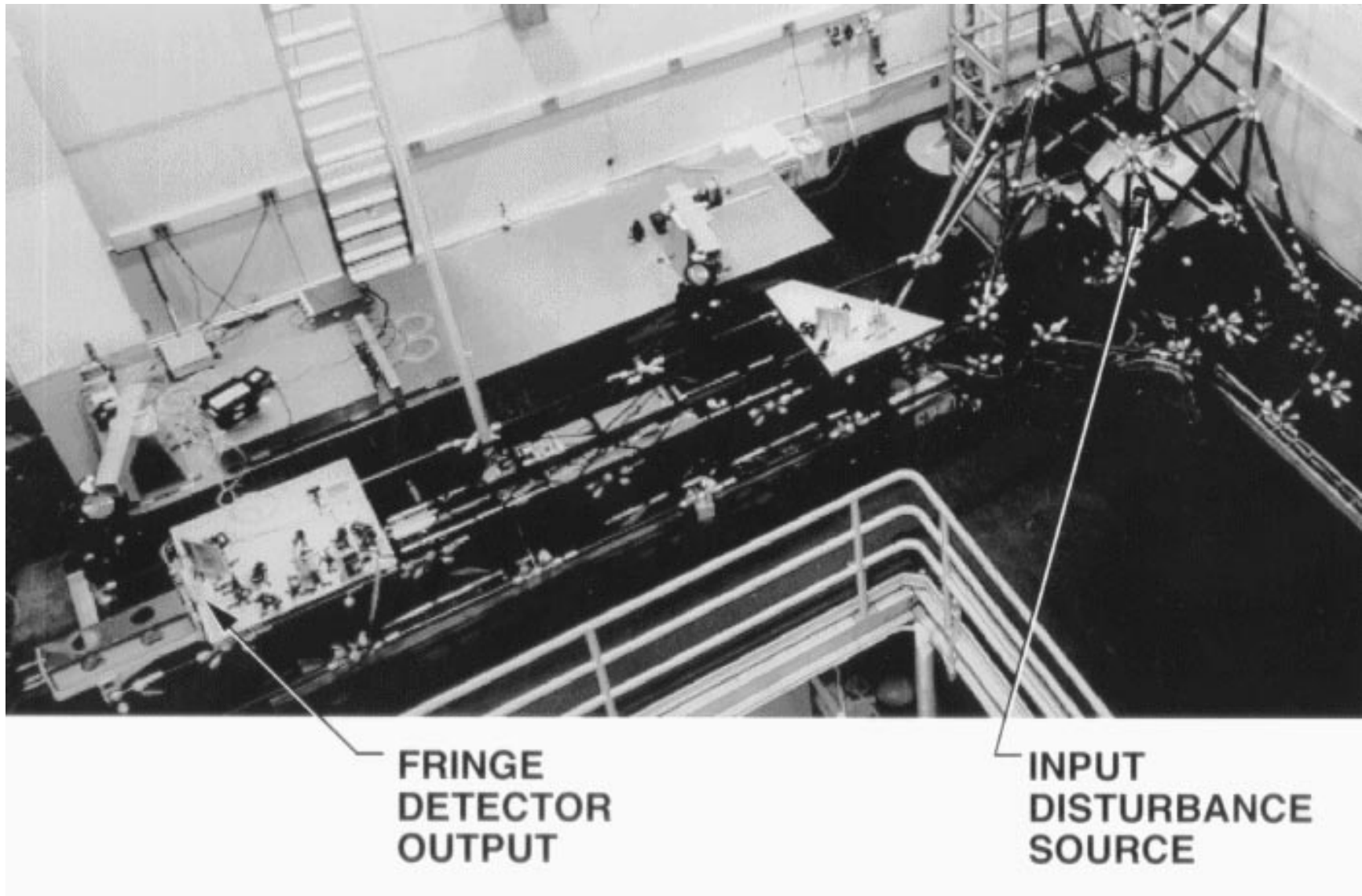
with the subscript  $k$  referring to the set of 622 kept modes.

- Incorporate linear optical model:

$$y_{opt} = C_{opt} C_d x \Rightarrow \begin{cases} \dot{x} = Ax + Bf \\ y_{opt} = Cx \end{cases}$$

- Resultant model is amenable to analysis with existing Matlab functions.
  - Input: forces at disturbance location
  - Output: stellar optical pathlength difference





- Typically, disturbance has broadband PSD,  $\Phi_d(\omega)$ , and the performance measure is OPD variation,  $\sigma_{opd}$  :

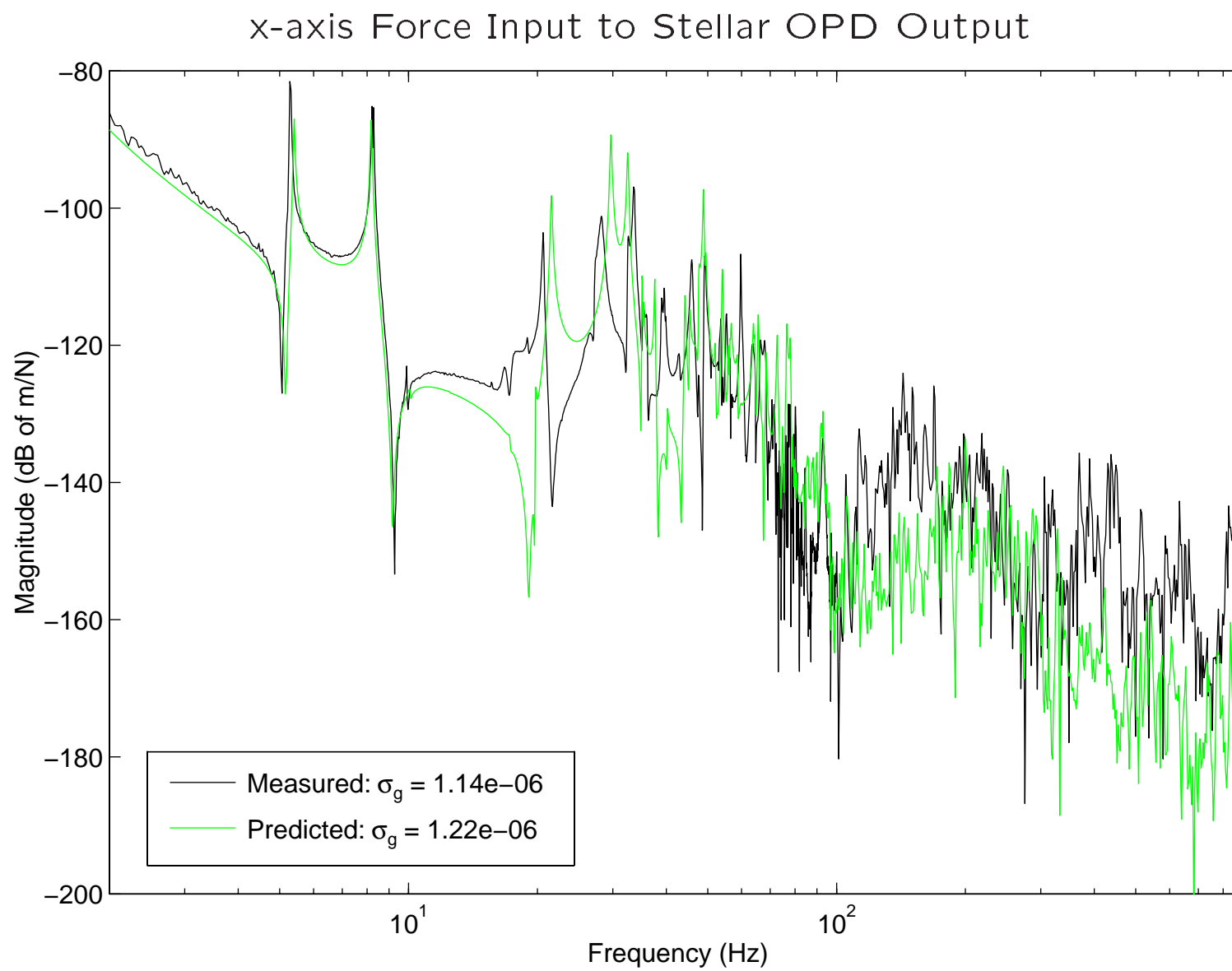
$$\sigma_{opd}^2 = \frac{1}{\pi} \int_0^\infty |G(j\omega)|^2 \Phi_d(\omega) d\omega$$

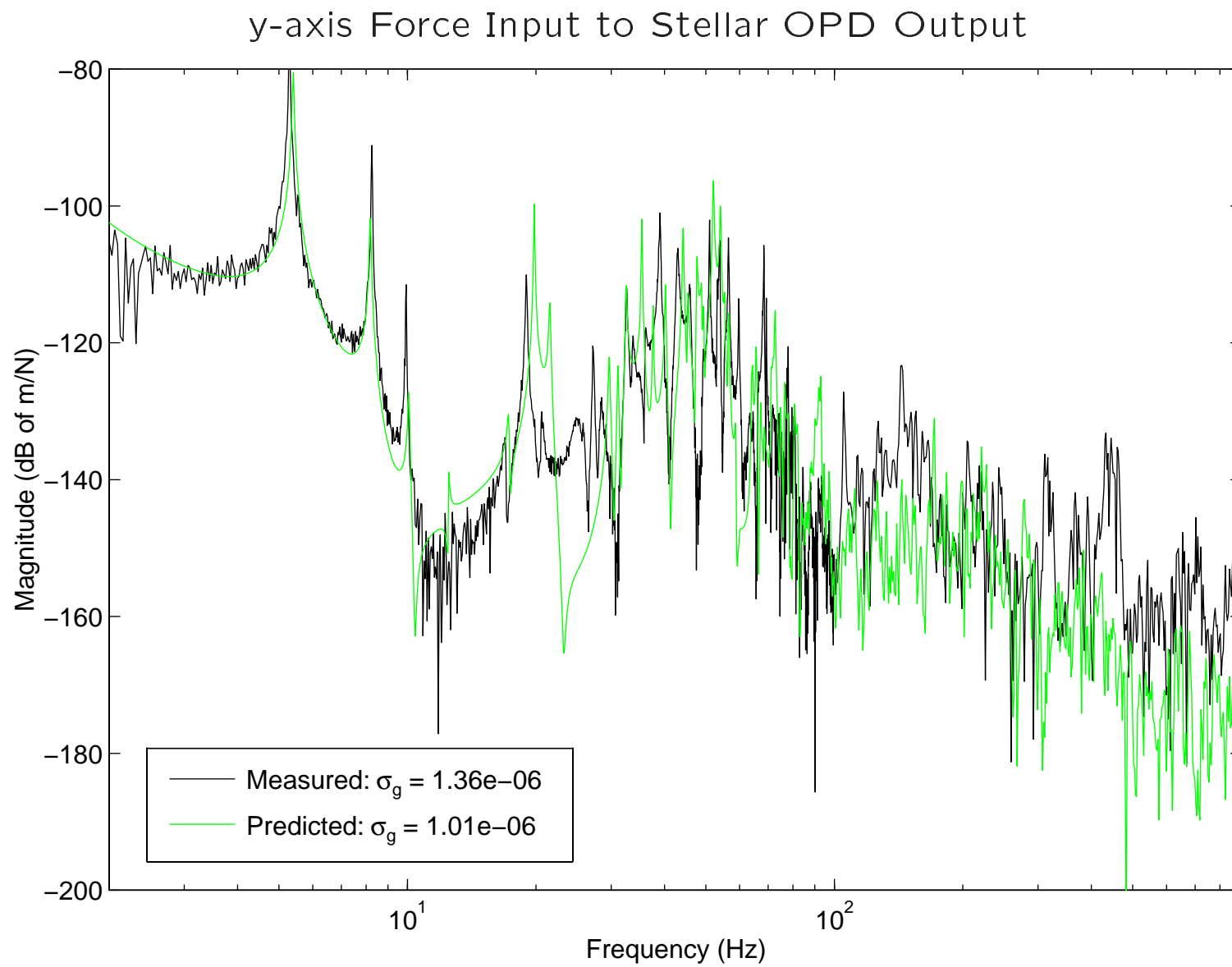
- Generally, an accuracy of a **factor of two** in  $\sigma_{opd}$  is desired.
- Use a bandlimited white noise disturbance to characterize the accuracy of the predicted transfer functions in the frequency range of interest ( $[\omega_1, \omega_2]$ ):

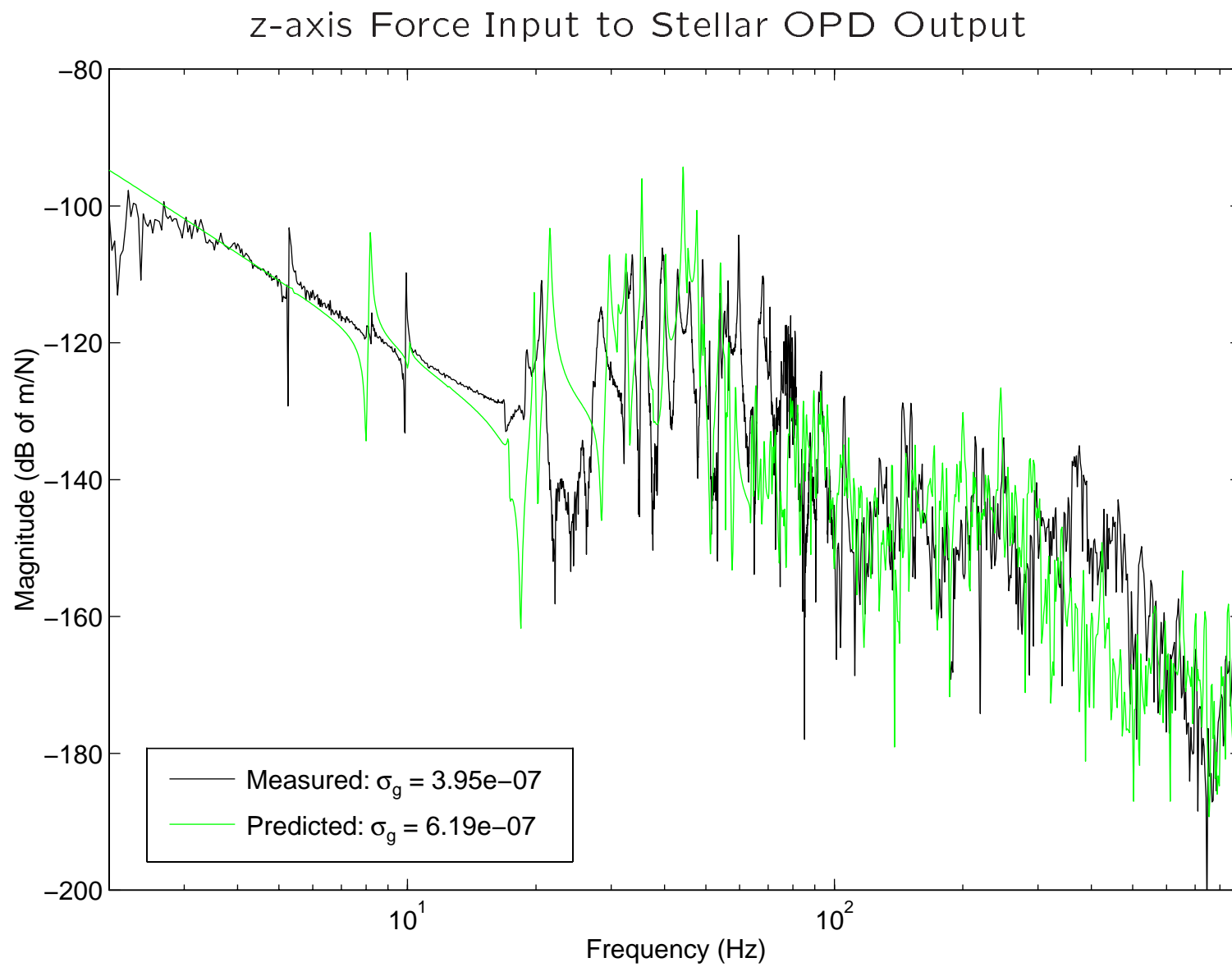
$$\sigma_g^2 = \frac{1}{\pi} \int_{\omega_1}^{\omega_2} |G(j\omega)|^2 d\omega$$

- Apply the factor of two desirment to the ratio of  $\sigma_g$  for the predicted and measured transfer functions:

$$\frac{1}{2} \leq \frac{\sigma_{gp}}{\sigma_{gm}} \leq 2$$







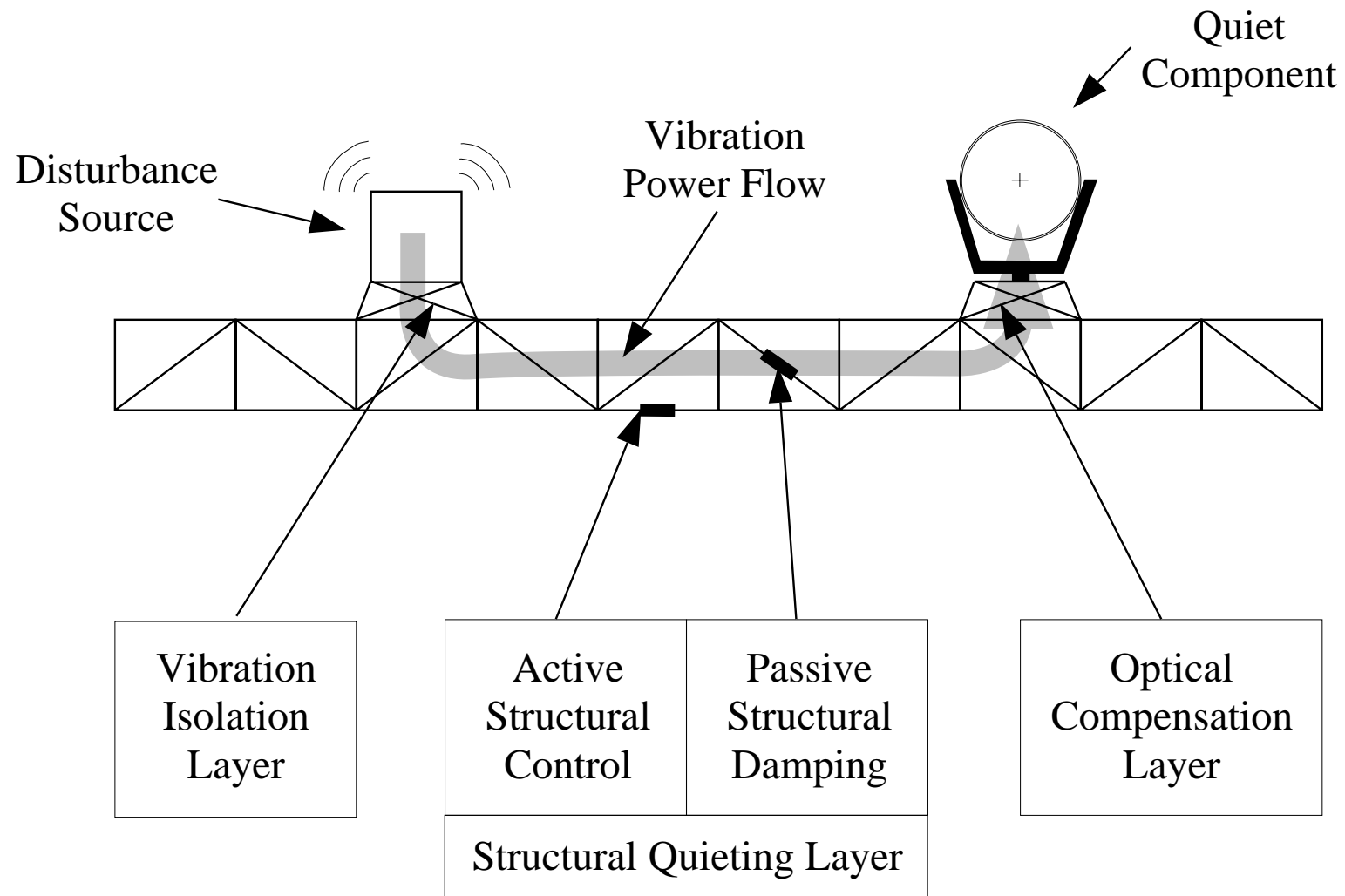
Disturbance Input		$\sigma_g$			
		4 - 10 Hz	10 - 100 Hz	100 - 900 Hz	4 - 900 Hz
x-axis Force	measured	997	541	70	1,137
	predicted	666	1,025	22	1,223
	factor	0.67	1.89	0.32	1.08
y-axis Force	measured	1,313	360	69	1,363
	predicted	864	522	24	1,010
	factor	0.66	1.45	0.35	0.74
z-axis Force	measured	185	346	50	395
	predicted	177	591	47	619
	factor	0.95	1.71	0.95	1.57

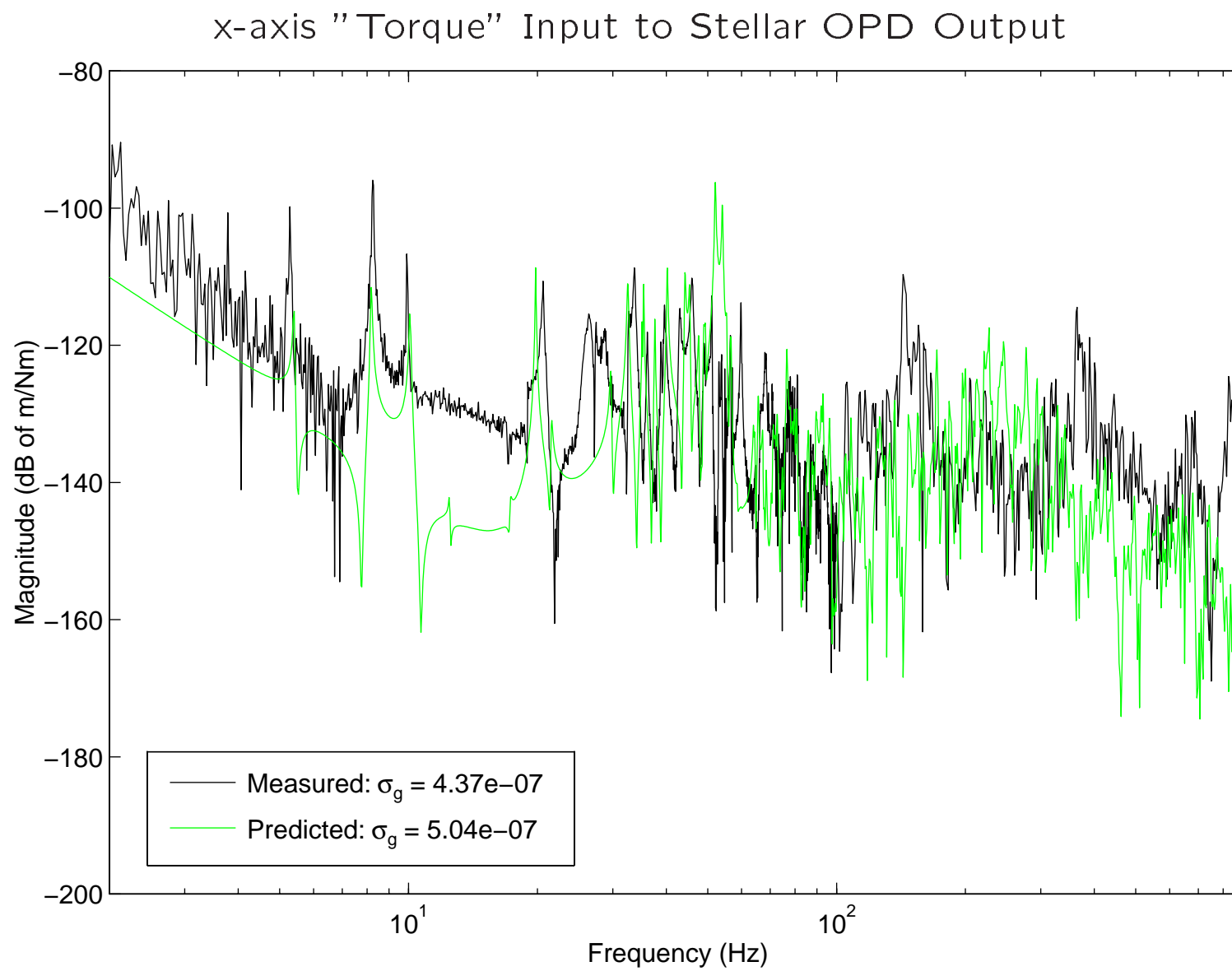
Note: units are absent since the *separate* values are not meaningful. It is the *ratio* that is significant.

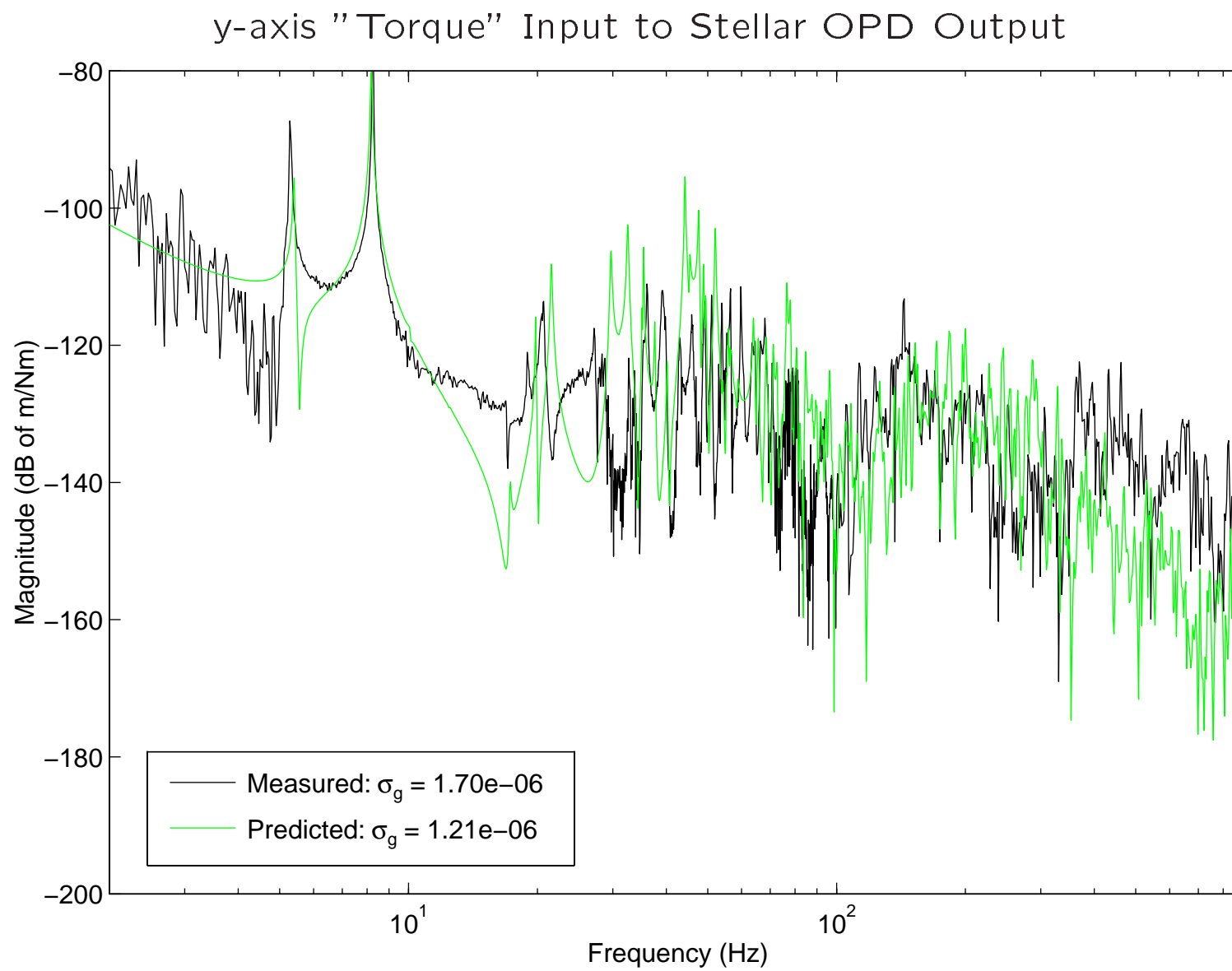
- Validate disturbance torque transfer functions
  - Torques not yet included because of bad measurement approach: torques generated by tandem force shakers.
  - Measurement will be improved by fabrication of torque shaker.
- Validate for various CSI vibration attenuation layers
  - Active optics (draft submitted to ACC 97)
  - Active optics and isolation
- Determine, in retrospect, how much parameter identification and/or model fidelity is needed for valid model.
  - Simple beam model of structure.
  - Rod model of truss structure.
  - Structural model before various parameter identification.
- Time domain validation for particular RWA disturbance input.
  - In lab, generate RWA disturbance for several wheel speeds and measure resultant OPD. Compare with predicted OPD.
  - Combines validation and performance prediction efforts.

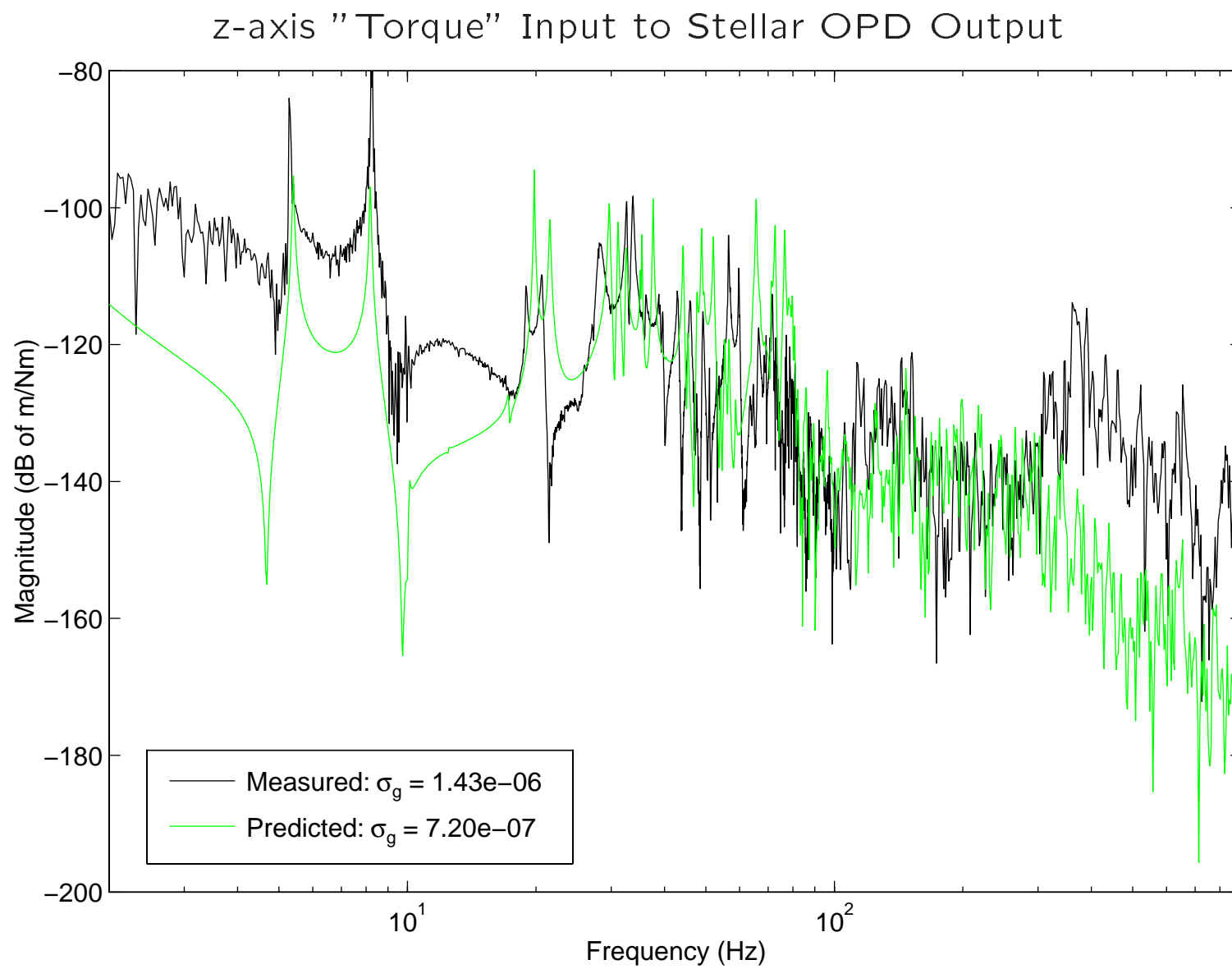


# Supporting Vugraphs









Disturbance Input		$\sigma_g$			
		4 - 10 Hz	10 - 100 Hz	100 - 900 Hz	4 - 900 Hz
x-axis Torque	measured	196	201	335	437
	predicted	44	471	175	504
	factor	0.23	2.34	0.52	1.15
y-axis Torque	measured	1,667	201	241	1,697
	predicted	1,065	542	199	1,212
	factor	0.64	2.70	0.82	0.71
z-axis Torque	measured	1,292	499	349	1,429
	predicted	219	682	73	720
	factor	0.17	1.37	0.21	0.50

Note: units are absent since the *separate* values are not meaningful. It is the *ratio* that is significant.